## **Strength of Mechanical Constructions**

## Energy methods in mechanics

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#### Basic considerations

- Equivalence of work and energy
- Elastic strain energy for simple loads



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#### Solving mechanical problems

There are two approaches to solve problems in mechanics

- vectorial approach
  - equation of equilibrium is written for vectorial quantities like stress or forces; from this equations the relation between the quantities is derived
- energy approach
  - internal energy of the system approach minimum value at equilibrium; by minimisation of the expression for energy one can find the equilibrium of the structure
- energy of mechanical systems is defined through the work done on these systems

# Equivalence of work and energy Definition of work

- let's start with the definition of work
- the work done by force *F* on the point by moving it from position *a* to position *a*<sub>1</sub> is the product of the force and the displacement of the point
- since in general case the direction of force does not correspond with the direction of displacement we can write

$$W = F \cdot s \cos \alpha$$
 or  $W = F \cdot s_F$ 

where  $s_F$  is the projection of displacement on the direction of the force

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- in general case the force and the displacement depends on time
- having the relation of force and displacement in the form of a plot, one may determine the work by calculating the area under the plot
- elementary work will be equal to

$$dW = F(t) \cdot ds_F(t)$$

• and after integration

$$W = \int\limits_{0}^{s_F} F(t) \cdot ds_F(t)$$



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How the potential energy of load is transformed into energy of elastic deformation?

Assumptions:

- the load is a static one is applied gradually and accelerations of structure can be omitted
- the deformation of the bar is slow enough to omit the inertia phenomenon
- at each point of loading the system is and its parts are in equilibrium
- material of the system follows the Hooke's law

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Transformation of the potential energy into energy of elastic deformation

- as an example let's consider the vertical bar fixed at the upper end with a rigid disc fixed at the other end
- next to the bar there is a weight *Q*, which will be placed on the disk
- after elevation the weight get some energy and after dropping on the disk it will lower because of the elongation of the bar



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Transformation of the potential energy into energy of elastic deformation

- lowering of the weight about  $\Delta l$  decreases its energy about  $U_Q$
- this energy is transformed into energy of elastic deformation  $U_{arepsilon}$
- both energies can be expressed by the work – energy of the weight as the work of the external forces  $(W_e)$ , and the energy of elastic deformation as a work of internal forces  $(-W_i)$
- which is

$$U_Q = W_e$$
 and  $U_arepsilon = -W_i$ 



Transformation of the potential energy into energy of elastic deformation

- it should be explained why the work of internal forces is negative
- internal forces are the forces between atoms
- during elongation the atoms moves according to the axis of the bar but the forces are directed in the opposite way

   thus the work is negative



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• according to the law of conservation of energy for elastic systems

$$U_{\varepsilon} = U_Q$$

 $U_{c} = W_{a}$ 

• finally, because 
$$U_Q = W_e$$

The above coincides with the principle of virtual work

$$W_e = -W_i \quad o \quad W_e + W_i = 0$$

#### Attention!

only a different form of energy can change into potential energy of elastic deformation; the work  $W_e$  is only a measure of the potential energy of the external forces

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#### First law of thermodynamics

- one can not measure the total amount of internal energy of a system
- however, one can measure the changes of this energy

#### First law of thermodynamics

The work performed on a mechanical system by external forces plus the heat that flows into the system from the outside equals the increase in internal energy plus the increase in kinetic energy

 $\delta W + \delta H = \delta U + \delta K$ 

• assuming the lack of heat exchange and the movement

$$\delta W = \delta U$$

Determination of elastic strain energy for simple case of load.

- tension (compression)
  - load: axial force Fdisplacement: elongation  $\Delta l$

$$U_F = rac{1}{2} rac{F^2 l}{EA}$$
 $U_F = rac{1}{2} \int \limits_0^l rac{F^2 dx}{EA} \quad or \quad U_F = \int \limits_0^l rac{EA}{2} \left(rac{du}{dx}
ight)^2 dx$ 

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Determination of elastic strain energy for simple case of load.

- torsion
  - load: torsional moment T displacement: angle of twist  $\varphi$

$$U_T=rac{1}{2}rac{T^2l}{GI_0}$$
 $U_T=rac{1}{2}\int\limits_0^lrac{T^2dx}{GI_0} \quad or \quad U_T=\int\limits_0^lrac{GI_0}{2}\left(rac{darphi}{dx}
ight)^2dx$ 

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Determination of elastic strain energy for simple case of load.

- bending
  - load: bending moment M displacement: angle of deflection  $\theta$

$$U_M=rac{1}{2}rac{M^2l}{EI}$$
 $U_M=rac{1}{2}\int\limits_0^lrac{M^2dx}{EI}\quad or\quad U_M=\int\limits_0^lrac{EI}{2}\left(rac{d^2v}{dx^2}
ight)^2dx$ 

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- all above formulae have the same structure, which helps to remember them and the use for solving problems
- the structure is

$$U = rac{P_i \delta_i}{2}$$

•  $\delta_i$  – generalised displacement

• 
$$u, \varphi, \theta$$

- $P_i$  generalised force
  - F, T, M
- we will return to the above equation when the Castigliano's theorem will be derived

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#### Energy of elastic deformation Example 1

## Determine the energy of elastic deformation stored in a simply supported beam loaded with transverse force F.



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#### Energy of elastic deformation Example 2

Determine the energy of elastic deformation stored in a cantilever beam loaded at one end with transverse force F and bending moment M.



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- consider a simply supported beam loaded with two steps
- energy of elastic deformation do not depends on the order of these load steps



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#### Theorem

The partial derivative of the strain energy of a structure with respect to any load is equal to the displacement corresponding to that load.

$$\delta_i = rac{\partial U}{\partial P_i}$$

δ<sub>i</sub> - generalised coordinate
u, φ, θ
P<sub>i</sub> - generalised force
F, T, M

For convenience a modified Castigliano's theorem can be used

$$\delta_i = rac{\partial U}{\partial P_i} = rac{\partial}{\partial P_i} \int rac{M(x)^2}{2EI} dx = \int rac{M(x)}{EI} rac{\partial M(x)}{\partial P_i} dx$$

Since there are usually two generalised coordinates we can write

$$u = rac{\partial U}{\partial F} = \int rac{M(x)}{EI} rac{\partial M(x)}{\partial F} dx$$
 $heta = rac{\partial U}{\partial M} = \int rac{M(x)}{EI} rac{\partial M(x)}{\partial M} dx$ 

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Determine the vertical displacement of the free end of the cantilever beam.



# Castigliano's theorem Example

Determine horizontal u and vertical v displacement of the point where the load is applied.



- special case of Castigliano's theorem is Castigliano-Menabrea theorem
- it gives the possibility to determine statically indeterminate problems
- since it is known that the displacement of the point at the support is equal to zero one can write:



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Castigliano's theorem gives the possibility to:

- determine the displacement of points of a structure at which load is applied; even complicated structures can be easily analysed
- determine the displacement of any point of the structure by applying a fictitious load
- solve statically indeterminate problems

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#### Castigliano's theorem Analysis of frames



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The frame is loaded with load intensity q. Both elements of the frame are of equal length l and equal stiffness E, I. Determine:

- maximum bending moment
- displacement of point C
- rotation at point B

